[Monetary stamp of 3,000 Yen]



UTILITY MODEL APPLICATION (20) SHOWA 50 [1975] September 25

To: The Minister of Patent office

- 1. TITLE OF THE UTILITY MODEL
 Inlet valve of refrigeration compressor [Reito asshukuki no kyuunyuuben]
- 2. Inventors of the Utility Model Address: c/o Matsushita Denki Sangyo Kabushiki Kaisha [Japanese Company or Corporation], 1006-banchi, Kadoma, Oaza, Kadoma-shi, Osaka-fu

Name: Hiroshi MOROKOSHI

Name: Jiro YUUDA Name: Michimasa HORI Name: Mitsuhiro IKOMA Name: Takeshi AIZAWA

3 Assignee of the Utility Model

Addres: 1006 -banchi, Kadoma, Oaza, Kadoma-shi, Osaka-fu

Name: (582) Matsushita Denki Sangyo Kabushiki Kaisha Representative: Masaharu MATSUSHITA

3. Agents \mp 571

Address: c/o Matsushita Denki Sangyo Kabushiki Kaisha 1006-banchi, Kadoma, Oaza, Kadoma-shi, Osaka-fu

Name: (5971) Toshio NAKAO, patent agent [stamp of Nakao]

Name: (6152) Shigetaka KURINO, patent agent

[Contact Tel. No. (Tokyo) 453-3111, Patent Section]

5. Index of Attached Documents

Specification
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50-131611

[Amendments: There are no amendments attached to this Utility Model, Translator's note]

[Note: All names, addresses, company names, and brand names are translated in the most common manner. Japanese language does not have singular or plural words unless otherwise specified with numeral prefix or general form of plurality suffix. Translator's note]

SPECIFICATION

1. TITLE OF THE UTILITY MODEL Inlet valve of refrigeration compressor

2. CLAIMS

An inlet valve of refrigeration compressor is equipped with a supporting end that supports to a frame of a cylinder of a refrigeration compressor, and a top end that touches with a valve lift arranged at said cylinder, and it is formed in a way so the plate thickness within a range from above-described support end to the top end would become thinner as it approaches toward top end.

3. DETAILED DESCRIPTION OF THE UTILITY MODEL

This utility model relates to an inlet valve of refrigeration compressor; and it offers inlet valve that is strong in strength and above all, shows a quick valve action with light weight.\

The general internal structure at nearby inlet valve of refrigeration compressor shows a structure that is illustrated in the Figure 1; and during the process of inlet, coolant is given from an inlet hole (2) of the valve plate (1) and this coolant enters cylinder (5) through opening of inlet valve (4) that is supported at the frame (3). At this time, the inlet valve (4) repeats chattering during the process of inlet in which the inlet valve (4) strikes valve lift (6) of the frame (3) and receives impact to spring back to the valve plate (1) side, and it receives an inlet coolant pressure again to strike against valve lift (6).

Because both planes of the inlet valve (4) have been generally set as parallel with constant plate thickness (t) as it is illustrated in the Figure 2b, during repeat chattering in the process of inlet, a large pressure is applied to the supporting end (7) that supports the frame (3) to result in breakage, or when plate thickness (t) of the inlet valve (4) is increased in order to prevent from such breakage, rigidity of the vale becomes large to cause delayed opening or closing, or furthermore, as the thickness of plate thickness (t) of the inlet valve (4) is increased more, impact when striking against valve lift (6) becomes large to display such defects as becoming the cause for breakdown, cracks and the like.

In addition, although it is necessary to reduce a top clearance of the cylinder in order to improve the performance of refrigeration compressor, in ordinary case, because top end of inlet valve (4) and piston (8) become easy to contact when top clearance of the cylinder is reduced, it is not possible to reduce the top clearance of the cylinder; and this has been regarded as a significantly large problematic issue from the standpoint of improving a capability of refrigeration compressor.

This utility model resolves above-described conventional defects; and the inlet valve by the example of this utility model is explained below with reference to the Figures 3 a \sim c.

The Figures 3 a \sim c illustrate side-plane views of the inlet valve by the example of this utility model; and hole (10) to attach this inlet valve to the frame (3) is arranged at the supporting end (9) that is supported with the frame (3).

The inlet valve of the first example that is illustrated in the Figure 3 a has an angle (α) at the plane (12) of the cylinder side against the plane (11) that touches valve plate (1); and in addition, the inlet valve of the second example that is illustrated in the Figure 3b of which plane (12) against the plane (11) has an angle (β) that is greater than said angle (α). Both are set to show thinner plate thickness on the top end (13) than that of the supporting end (9); and through this, stress distribution of the inlet valve during process of inlet becomes uniform at any portions to eliminate chance for breakage of the supporting end (9) caused by concentration of the stress at the supporting end (9) as in the conventional case. In addition, because top end (13) of the inlet valve is thin and light, impact on the valve lift (6) becomes small to eliminate breakdown or cracks and the like, ant at the same time, valve action becomes fast to cause no delays in closing or opening. Above all, because top end (13) of the inlet valve being thin, there would be no danger of piston (8) coming in contact with the inlet valve even if top clearance of the cylinder is reduced, it is possible to reduce the top clearance, and is possible to improve the capability of the refrigeration compressor through this. The inlet valve of the third example that is illustrated in the Figure 3c shows the planes of each plane (11) that touches the valve plate (1) and plane (12) of the cylinder side being in a wedge-shape with incline provided; and it is possible to attain the same effect as the inlet valve of the first example and the inlet valve of the second example described previously.

Furthermore, although this utility model eliminates conventional defects of the inlet valve, it is all right to use this utility model as outlet valve as well.

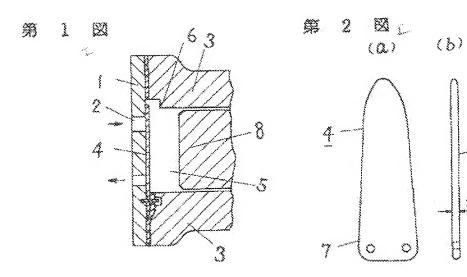
As described above, according to this utility model, it is possible to improve the performance of the refrigeration compressor; and above all, it offers the inlet valve of refrigeration compressor that displays very excellent effect of eliminating breakage of inlet valve that is arranged within a cylinder of refrigeration compressor and allows fast valve movement without delays in closing or opening.

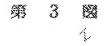
4. BRIEF DESCRIPTION OF THE FIGURES

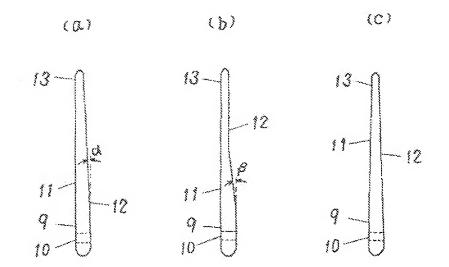
Figure 1 illustrates a cross-sectional view that shows the main structure of a general refrigeration compressor; and Figure 2a illustrates a frontal view that shows a shape of inlet valve of conventional refrigeration compressor; and Figure 2b illustrates its side-plane view; and Figures 3 a \sim c illustrate side-plane views of the shape of inlet valve of refrigeration compressor by the examples of this utility model.

3: frame, 6: valve lift, 9: supporting end, 13: top end

Figures 1 through 3 [I: Figure]







公開実用 昭和52— 44403



実用新案登録願(20)

昭和 50年 9月25日

特許庁長官殿

1考案の名称

レイトウアッシュタキ キュウニュウベトウ 優 任 網 優 の 吸 入 并

2 考 案 者

氏 名

Ť

(ほか4名)

3 実用新案登録出願人

称

大阪府門真市大字門真1006番地(582)松下電器產業株式会社

代表者

名

松

下

正 治

4代理人〒571

住 所

大阪府門真市大字門真1006番地

松下電器産業株式会社内

氏 名

(5971) 弁理士 中尾 敏 ! (ほか 1名)

〔連絡先 電話(東京)453-3111 特許分室〕

5 添付書類の目録

- (1) 明 細 書
- (2) 図 面
- (3) 委任 状
- (4) 願書副本



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郡

1、考案の名称

帝漢圧纏機の吸入弁

例

2、 奥 用 新 楽 登 録 請 求 の 軽 囲

冷機圧離機のシリンダのフレームに支持させる 支持機と、前配シリンダに設けた弁リフトに当盤 する先端とを係え、かつ上記支持嫌から先端へか けての複厚を先端へ近づく程準くなるように形成 してなる冷使圧縮機の吸入弁。

3、考察の辞細な説明

本考案は冷凍圧縮機の吸入弁に関するものであり、強度的に強固で、しかも軽量で弁動作のすば やい吸入弁を提供するものである。

一般的な冷凍圧縮機の吸入弁付近の内部構造は 第1回に示すような構造をしており、吸入行程中 は弁板1の吸入孔2から冷媒が吸入され、フレームの なるで支持させた吸入弁4が防いてシリング内の から、吸入弁4はフレーム3の 分のである。との時、吸入弁4はフレーム3の 分のである。との時、吸入弁4はフレーム3の 分のである。を受け 弁板1億にはねる り、再び吸入冷離圧力を受け、弁リフトのに当る



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というように、数入行程中数入弁4はテャタリンクを繰り返している。

また。僧僚圧解機の性能を同上させるためにはシリンダのトップクリアランスを小さくしなブクリアランスを小さくかのトップクリアランスを小さくするとめ入弁4先瀬とピストンのとが優別しやすくなるため。シリンダのトップクリアランスを小さくすることができないたりの発圧を機の能力を同上させる上で非常に大きな問題となっていた。



本考案は上述のような従来の欠点を解補するもので、以下第3回a~cを参照したがら本考案の実施例による吸入弁を説明する。
第3回a~cは本考案の実施例による吸入弁の側面図で、フレーム3に支持させる支持端9にはと

面図で、フレーム3に支持させる支持端9にはとの吸入弁をフレーム3に取り付けるための孔1のが設けられている。

第3図&に示す第1の実施例の吸入弁は、弁板



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はやくなり、別じ避れや開き遅れがかくかる。しから、数入弁の先端13があいってもピンクリンスをいてもピンクが、サークリアランスを映性がないため、シリンをを映性がないない。とかできたが、アクリアをではないが、から、第3のであるのであり、前述した第1のであり、前述したので、第4のの数人弁と同様の効果を得るととができる。

たか。本考案は吸入弁の従来の欠点を解摘するものであるが。吐出弁に本考案を用いてもよい。

以上のように本考察によれば、冷凍圧解機の性能を同上させることができ、しかも冷凍圧網機のシリング内に散けている吸入弁の折損をなくし、かつ閉じ遅れや講き遅れがないというすばやい弁動作を行なえるという非常に優れた効果を有する冷凍圧組織の吸入弁を提供するものである。

4。 凶面の簡単な説例

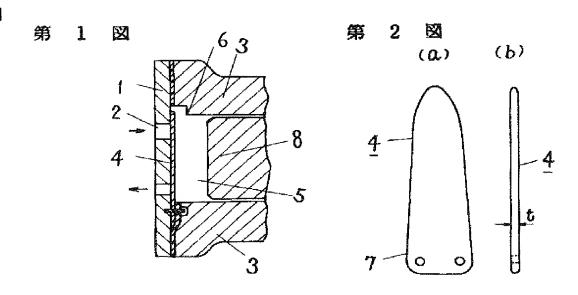
第1 図は一般的な俗様任難機の要部構造を示す断面図。第2 図 a は従来の冷様圧縮機の吸入弁の形状を示す正面図。第2 図 b はその側面図。第3 図 a ~ c は本考案の実施例による冷凍圧縮機の吸入弁の形状を示す舞面図である。

3 … … フレーム。 6 … … 弁 リフト。 9 … … 支持端。 1 3 … … 先端。

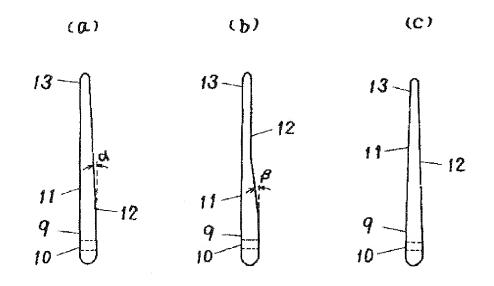
代理人の氏名 弁理士 中 岸 骸 男 ほか1名



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第 3 図



でを入りたも お理士 中 尾 徳 男 ほか1名

前記以外の考案者および代理人

(1) 考案者

-,•

大阪府門真市大字門真1006番单 住 所 松下電器產業株式会社內 氏 名 所 所 住 间 溢 Æ 名 住 所 口 所 彩 1 氏 名 住 所 同 所 ザワバ 氏 名

(2) 代理人

大阪府門真市大字門真1006番地 住 Ħ 松下電器產業株式会社內 \mathbb{R} 名

(6152) 弁理士 重 孝

